

# Calculation with $\LaTeX$ by means of Calc $\TeX$

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## 1 Calculation with $\LaTeX$

This folder consider couples of examples – all files `*-calc.tex` in  $\LaTeX$  input format with defined operations that are included in a single pdf document with calculated operations by means of Calc $\TeX$  package.

For received a final calculation of all `*-calc.tex` files please use a LINUX prompt command: `sh go`.

For more info please visit a web page on: <http://sg.bzip.pl/CalcTeX> or contact me by e-mail: CalcTeX (at) onet (dot) eu

I am open for any kind of questions or commands.

The file `"1st-example-ke-eng-calc.tex"` is the simplest and I recommend this file for study of Calc $\TeX$  used at first.

The file `"http://sg.bzip.pl/CalcTeX/examples/all-in-one.tgz"` consider all necessary files for calculation of this example if you have installed a  $\LaTeX$  compiler and python language as well it's checked for LINUX system.

All files which suits to mask `*-iso-calc.tex` are autamaticlly calculated and included

into single pdf format document in order to `ls *-calc.tex` comment and for calculation are included all python files available on `bin/py` folder.

## 1.1 Kinetic energy

An  $m_a := 0.725 \cdot \text{ton}$  automobile has a kinetic energy of  $E_k := 145 \cdot \text{kJ}$  as it travels along a highway. What is the car's speed?

### 1.1.1 Calculation Kinetic energy

List the given and unknown values.

**Given:** mass,  $m_a \cdot \text{kg}^{-1} = 725.0$

kinetic energy,  $E_k \cdot \text{MJ}^{-1} = 0.145$

**Unknown:** speed,  $v = ? \text{ m/s}$

$$E_k := \frac{m_a \cdot v^2}{2} \Leftrightarrow v := \sqrt{\frac{2 \cdot E_k}{m_a}}; \quad v \cdot \left(\frac{\text{m}}{\text{s}}\right)^{-1} = 20.0 \quad (1)$$

The car's speed is  $v \cdot (\text{m/s})^{-1} = 20.0$ , that is equivalent to  $v \cdot (\text{km/hr})^{-1} = 72.0$ .

### 1.1.2 Source – 1st-ke-eng-iso-calc.tex

```
\Task{Kinetic energy}
```

```
An $m_a:=0.725\cdot \text{ton}$ automobile has a kinetic energy of
$E_k:=145\cdot \text{kJ}$ as it travels along a highway.
What is the car's speed?
```

```
\Calculation{Kinetic energy}
```

```
List the given and unknown values.\
{\bf Given:} mass, $m_a\cdot \text{kg}^{-1}$\
kinetic energy, $E_k\cdot \text{MJ}^{-1}$\
{\bf Unknown:} speed, "$v = ? \text{ m/s}$"
```

```
\begin{equation}"
E_k:=\frac{m_a\cdot v^2}{2}
\;\;\;\;\;\Leftrightarrow\;\;\;\;\;
v:=\sqrt{\frac{2\cdot E_k}{m_a}}
"\;\;\;\;\;
v\cdot \left(\frac{\text{m}}{\text{s}}\right)^{-1}
\end{equation}
```

```
The car's speed is
$v\cdot \left(\frac{\text{m}}{\text{s}}\right)^{-1}$" that is equivalent to
$v\cdot \left(\frac{\text{km}}{\text{hr}}\right)^{-1}$"
```

This following example based on calculation available on <http://blowers.chee.arizona.edu/cooking/heat/ex11.html>

## 1.2 Water/Bucket system

There is  $V_w := 6 \cdot \text{l}$  (liters) of water ( $\rho_w := 998 \cdot \text{kg/m}^3$ ) in a  $m_b := 750 \cdot \text{gm}$  bucket. The initial temperatures of the water and bucket are  $T_w := 60 \cdot \text{°C}$  and  $T_b := 22 \cdot \text{°C}$  respectively. The heat capacity of water is  $c_w := 4.2 \cdot \text{kJ}/(\text{kg} \cdot \text{°C})$  and the heat capacity of the bucket is  $c_b := 910 \cdot \text{J}/(\text{kg} \cdot \text{°C})$ . Determine the final temperature of the water/bucket system.

### 1.2.1 Calculation Water/Bucket system

First we must determine what is going on in this problem. There is heat that is being transferred. The heat moves from the water to the bucket since the water is at a higher temperature and heat flows from hot to cold. The temperature of the water will decrease as more heat is transferred while the temperature of the bucket will increase as more heat is transferred. Eventually, the water and the bucket will reach the same final temperature. Even though heat is still being transferred, it is transferred equally now, from the water to the bucket and vice versa. In order to determine this final temperature, the following equation must be used:

$$\Delta Q_s = m_s \cdot c_s \cdot \Delta T \quad (2)$$

Recall that  $\Delta Q_s$  is the amount of heat lost or gained. We want to know the final temperature of the system, and we have determined that occurs when the amount of heat transferred from the water to the bucket equals the amount of heat that is transferred from the bucket to the water. We will set  $\Delta Q_s$  for the water to the bucket equal to  $\Delta Q_s$  for the bucket to the water.

$$-\Delta Q_w = \Delta Q_b \quad (3)$$

where:

$\Delta Q_w$  heat transferred from the water to the bucket,

$\Delta Q_b$  heat transferred from the bucket to the water.

Since heat is lost from the water and gained by the bucket, the sign for  $\Delta Q_w$  is negative and the sign for  $\Delta Q_b$  is positive. Plugging in the definition of  $\Delta Q_s$  gives the following equation

$$-m_w \cdot c_w \cdot (T_f - T_w) = m_b \cdot c_b \cdot (T_f - T_b) \quad (4)$$

$$-m_w \cdot c_w \cdot T_f + m_w \cdot c_w \cdot T_w = m_b \cdot c_b \cdot T_f - m_b \cdot c_b \cdot T_b \quad (5)$$

$$+m_w \cdot c_w \cdot T_w + m_b \cdot c_b \cdot T_b = m_b \cdot c_b \cdot T_f + m_w \cdot c_w \cdot T_f \quad (6)$$

$$+m_w \cdot c_w \cdot T_w + m_b \cdot c_b \cdot T_b = T_f \cdot (m_b \cdot c_b + m_w \cdot c_w) \quad (7)$$

Mass of the water

$$m_w := \rho_w \cdot V_w; \quad m_w \cdot \text{kg}^{-1} = 5.988$$

where:

$m_w \cdot \text{kg}^{-1} = 5.988$	- mass of the water,
$c_w \cdot (\text{kJ}/(\text{kg} \cdot ^\circ\text{C}))^{-1} = 4.2$	- heat capacity of the water,
$m_b \cdot \text{kg}^{-1} = 0.75$	- mass of the bucket,
$c_b \cdot (\text{kJ}/(\text{kg} \cdot ^\circ\text{C}))^{-1} = 0.91$	- heat capacity of the bucket,
$T_w \cdot ^\circ\text{C}^{-1} = 60.0$	- initial temperature of the water,
$T_b \cdot ^\circ\text{C}^{-1} = 22.0$	- initial temperature of the bucket,
$T_f$	- the final temperature of the system.

All of the variables in this equation are known from the problem statement, except the final temperature ( $T_f$ ). This is the variable that we want to solve for. Solving the above equation for  $T_f$  gives

$$T_f := \frac{m_w \cdot c_w \cdot T_w + m_b \cdot c_b \cdot T_b}{m_w \cdot c_w + m_b \cdot c_b}; \quad T_f \cdot ^\circ\text{C}^{-1} = 58.996016584$$

Please note that in our problem we are using a temperature in °C and heat capacity in J/(kg · °C) generally is more safe to use a temperature in K. Additionally, the heat capacity have the same value in J/(kg · °C) and in J/(kg · K), for example:

$$\left( c_w \cdot \left( \frac{\text{kJ}}{\text{kg} \cdot ^\circ\text{C}} \right)^{-1} = 4.2 \right) = \left( c_w \cdot \left( \frac{\text{kJ}}{\text{kg} \cdot \text{K}} \right)^{-1} = 4.2 \right)$$

**Answer:** The final temperature of the water/bucket system is roughly  $T_f \cdot ^\circ\text{C}^{-1} = 59.0$ .

### 1.2.2 Source – 2nd-bucket-eng-iso-calc.tex

This following example based on calculation available on `\linkurl{http://blowers.chee.arizona.edu/cooking/heat/ex11.html}`{`http://blowers.chee.arizona.edu/cooking/heat/ex11.html`}

`\Task{Water/Bucket system}`

There is  $V_w=6$  (liters) of water  $(\rho_w=998 \text{ kg/m}^3)$  in a  $m_b=750$  bucket. The initial temperatures of the water and bucket are  $T_w := 60$  °C and  $T_b := 22$  °C respectively. The heat capacity of water is  $c_w= 4.2$  kJ/(kg·°C) and the heat capacity of the bucket is  $c_b= 910$  J/(kg·°C). Determine the final temperature of the water/bucket system.

`\Calculation{Water/Bucket system}`

First we must determine what is going on in this problem. There is heat that is being transferred. The heat moves from the water to the bucket since the water is at a higher temperature and heat flows from hot to cold. The temperature of the water will decrease as more heat is transferred while the temperature of the bucket will increase as more heat is transferred. Eventually, the water and the bucket will reach the same final temperature. Even though heat is still being transferred, it is transferred equally now, from the water to the bucket and vice versa. In order to determine this final temperature, the following equation must be used:

```
\begin{equation}
\Delta Q_s = m_s \cdot c_s \cdot \Delta T
\end{equation}
```

Recall that  $\Delta Q_s$  is the amount of heat lost or gained. We want to know the final temperature of the system, and we have determined that occurs when the amount of heat transferred from the water to the bucket equals the amount of heat that is transferred from the bucket to the water. We will set  $\Delta Q_s$  for the water to the bucket equal to  $\Delta Q_s$  for the bucket to the water.

```
\begin{equation}
-\Delta Q_w = \Delta Q_b
\end{equation}
```

where:

```
\
 $\Delta Q_w$  heat transferred from the water to the bucket,\
 $\Delta Q_b$  heat transferred from the bucket to the water.\
```

Since heat is lost from the water and gained by the bucket, the sign for  $\Delta Q_w$  is negative and the sign for  $\Delta Q_b$  is positive.

Plugging in the definition of  $\Delta Q_s$  gives the following equation

```

\begin{equation}"
-m_w \cdot c_w \cdot (T_f - T_w) = m_b \cdot c_b \cdot (T_f - T_b)
"\end{equation}
\begin{equation}"
-m_w \cdot c_w \cdot T_f + m_w \cdot c_w \cdot T_w =
m_b \cdot c_b \cdot T_f - m_b \cdot c_b \cdot T_b
"\end{equation}

\begin{equation}"
+ m_w \cdot c_w \cdot T_w + m_b \cdot c_b \cdot T_b =
m_b \cdot c_b \cdot T_f + m_w \cdot c_w \cdot T_f
"\end{equation}

\begin{equation}"
+ m_w \cdot c_w \cdot T_w + m_b \cdot c_b \cdot T_b =
T_f \cdot (m_b \cdot c_b + m_w \cdot c_w)
"\end{equation}

Mass of the water
\[
m_w = \rho_w \cdot V_w
";\;\;\;"
m_w \cdot \text{okg}
\]

where:\
\begin{tabular}{lcl}
$m_w \cdot \text{okg}$ & & --& mass of the water,\
$c_w \cdot \left( \frac{\text{kJ}}{\text{kg} \cdot \text{oC}} \right)^{-1}$ & & & --& heat capacity of the water,\
$m_b \cdot \text{okg}$ & & --& mass of the bucket,\
$c_b \cdot \left( \frac{\text{kJ}}{\text{kg} \cdot \text{oC}} \right)^{-1}$ & & & --& heat capacity of the bucket,\
$T_w \cdot \text{oC}$ & & --& initial temperature of the water,\
$T_b \cdot \text{oC}$ & & --& initial temperature of the bucket,\
"$T_f"$ & & --& the final temperature of the system.
\end{tabular}
\
All of the variables in this equation are known from the problem statement,
except the final temperature "$T_f$". This is the variable that we want to solve for.
Solving the above equation for "$T_f$" gives

\[ T_f := \frac{m_w \cdot c_w \cdot T_w + m_b \cdot c_b \cdot T_b}{m_w \cdot c_w + m_b \cdot c_b}
";\;\;\;"
T_f \cdot \text{oC} \]

Please note that in our problem we are using a temperature in
"$\text{oC}$" and heat capacity in "$\frac{\text{kJ}}{\text{kg} \cdot \text{oC}}$" generally is more
safe to use a temperature in K.
Additionally, the heat capacity have the same value in
"$\frac{\text{kJ}}{\text{kg} \cdot \text{oC}}$" and in "$\frac{\text{kJ}}{\text{kg} \cdot \text{K}}$",
for example:

\["\left(
c_w \cdot \left( \frac{\text{kJ}}{\text{kg} \cdot \text{oC}} \right)^{-1}
\right) = \left(
c_w \cdot \left( \frac{\text{kJ}}{\text{kg} \cdot \text{K}} \right)^{-1}
\right) \]
{\bf Answer:}
\texblanche{$T_f := \text{round}(T_f, 1)$}
The final temperature of the water/bucket system is roughly $T_f \cdot \text{oC}$."$

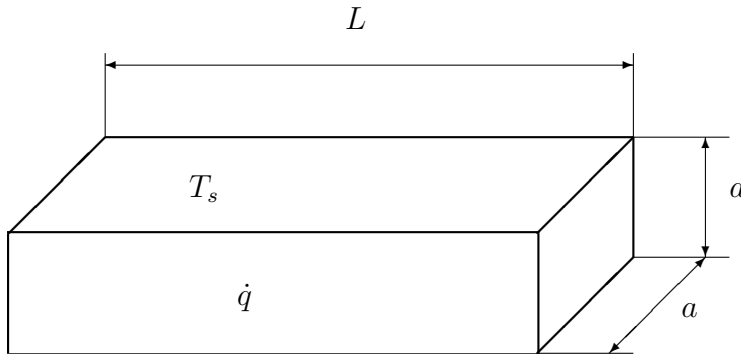
```

This following example based on Richard J. Pryputniewicz calculation available on <http://users.wpi.edu/~chs1t/courses/es3003/index.html>

### 1.3 PROBLEM 1. q1

Heat is generated in  $a := 2.5 \cdot \text{cm}$  square copper rod at the rate of  $\dot{q} := 35.3 \cdot \text{MW}/\text{m}^3$ . The rod is exposed to a convection environment at  $t_e := 20 \cdot ^\circ\text{C}$  where the convection heat transfer coefficient is  $h := 4 \cdot \text{kW}/(\text{m}^2 \cdot ^\circ\text{C})$ . Determine the surface temperature of the rod.

#### 1.3.1 Calculation



$$\dot{q} \cdot V = h \cdot A_c \cdot (t_s - t_e) \quad (8)$$

$$\dot{q} \cdot a^2 \cdot L = h \cdot 4 \cdot a \cdot L \cdot (t_s - t_e) \quad (9)$$

$$t_s := \frac{\dot{q} \cdot a}{4 \cdot h} + t_e; \quad t_s \cdot ^\circ\text{C}^{-1} = 75.15625 \quad (10)$$

#### 1.3.2 Source – 3rd-eng-calc.tex

```

This following example based on
\linkurl{mailto: rjp@wpi.edu} {Richard J. Pryputniewicz}
%http://users.wpi.edu/~chslt/rjp.html
calculation available on\
\linkurl{http://users.wpi.edu/~chslt/courses/es3003/index.html}{http://users.wpi.edu/~chslt/courses/es3003/index.html}

\Task{PROBLEM 1. q1}

Heat is generated in $a:= 2.5\cdot \text{cm}$ square copper rod
at the rate of $\dot{q}:=35.3\cdot \text{MW}/\text{m}^3$."$
The rod is exposed to a convection environment
at $t_e:=20\cdot \text{oC}$ where the convection
heat transfer coefficient is $h:=4 \cdot \text{kW}/(\text{m}^2\cdot \text{oC})$."$
Determine the surface temperature of the rod.

\Calculation{}

\input{figs/p1q1.latex}
\begin{equation}"
\dot{q} \cdot V = h \cdot A_c \cdot \left( t_s - t_e \right)
\label{p1q1e}"
\end{equation}

\begin{equation}"
\dot{q} \cdot a^2 \cdot L = h \cdot 4 \cdot a \cdot L \cdot \left( t_s - t_e \right)
\label{p1q1e}"
\end{equation}

\begin{equation}
t_s := \frac{\dot{q} \cdot a}{4 \cdot h} + t_e
";\;\;\;\;\;\label{p1q1e}"
t_s \cdot \text{oC}
\end{equation}

```

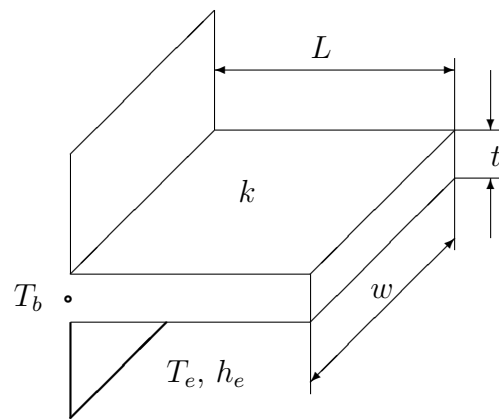
This following example based on Richard J. Pryputniewicz calculation available on <http://users.wpi.edu/~chslt/courses/es3003/index.html>

### 1.4 PROBLEM 4

A straight rectangular fin has a length of  $L := 3.5 \cdot \text{cm}$ , a thickness of  $t := 1.5 \cdot \text{mm}$ , and is made of a material characterized by thermal conductivity of  $k := 255 \cdot \text{W}/(\text{m} \cdot ^\circ\text{C})$ . The fin is exposed to a convection environment at  $t_e := 20 \cdot ^\circ\text{C}$  and  $h_c := 1.5 \cdot \text{kW}/(\text{m}^2 \cdot ^\circ\text{C})$ . Determine the maximum possible heat transfer from this fin for the base temperature of  $t_b := 150 \cdot ^\circ\text{C}$ . What is the actual heat transfer from this fin? What is its thermal efficiency?

#### 1.4.1 Calculation

$$\begin{aligned} L \cdot \text{cm}^{-1} &= 3.5 \\ t \cdot \text{cm}^{-1} &= 0.15 \\ w &:= 1 \cdot \text{m}^{-1} \\ k \cdot (\text{W}/(\text{m} \cdot ^\circ\text{C}))^{-1} &= 255.0 \\ t_e \cdot ^\circ\text{C}^{-1} &= 20.0 \\ h_c \cdot (\text{kW}/(\text{m}^2 \cdot ^\circ\text{C}))^{-1} &= 1.5 \\ Q_{max} &=? \\ t_b \cdot ^\circ\text{C}^{-1} &= 150.0 \\ Q_{act} &=? \\ \eta_{fin} &=? \end{aligned}$$



$$L_c := L + 0.5 \cdot t \quad (11)$$

$$A_c := 2 \cdot (w \cdot L_c); \quad A_c \cdot \text{m}^{-2} = 0.0715; \quad A_c \cdot \text{cm}^{-2} = 715.0 \quad (12)$$

$$Q_{max} := h_c \cdot A_c \cdot (t_b - t_e); \quad Q_{max} \cdot \text{kW}^{-1} = 13.9425 \quad (13)$$

$$A_m := L_c \cdot t; \quad A_m \cdot \text{m}^{-2} = 5.3625e - 05; \quad A_m \cdot \text{cm}^{-2} = 0.53625 \quad (14)$$

$$mL_c := \sqrt{\frac{2 \cdot h_c}{k \cdot A_m}} \cdot L_c^{1.5}; \quad mL_c = 3.16607321581 \quad (15)$$

$$\tanh(mL_c) = 0.996449887141 \quad (16)$$

Where

$$\tanh(x) = \frac{e^x - e^{-x}}{e^x + e^{-x}} \quad (17)$$

$$\eta_{fin} := \frac{\tanh(mL_c)}{mL_c}; \quad \eta_{fin} = 0.314727367064; \quad \eta_{fin} \cdot \%^{-1} = 31.4727367064 \quad (18)$$

$$Q_{act} := \eta_{fin} \cdot Q_{max}; \quad Q_{act} \cdot \text{kW}^{-1} = 4.38808631528 \quad (19)$$

## 1.4.2 Source – 4rd-eng-iso-calc.tex

```

This following example based on
\linkurl{mailto: rjp@wpi.edu} {Richard J. Pryputniewicz}
%http://users.wpi.edu/~chs1t/rjp.html
calculation available on\
\linkurl{http://users.wpi.edu/~chs1t/courses/es3003/index.html}{http://users.wpi.edu/~chs1t/courses/es3003/index.html}

\Task{PROBLEM 4}

A straight rectangular fin has a length of  $L=3.5 \text{ cm}$ ,
a thickness of  $t=1.5 \text{ mm}$ , and is made of a material
characterized by thermal conductivity of  $k=255 \text{ W/(m}\cdot\text{K)}$ .
The fin is exposed to a convection environment at  $t_e=20 \text{ }^\circ\text{C}$ 
and  $h_c = 1.5 \text{ kW/(m}^2\cdot\text{K)}$ .
Determine the maximum possible heat transfer from this
fin for the base temperature of  $t_b=150 \text{ }^\circ\text{C}$ .
What is the actual heat transfer from this fin? What is its thermal efficiency?

\Calculation{}

\begin{tabular}{l}
 $L \text{ cm}^{-1}$  \\
 $t \text{ cm}^{-1}$  \\
 $w = 1 \text{ cm}^{-1}$  \\
 $k \text{ W/(m}\cdot\text{K)}$  \\
 $t_e \text{ }^\circ\text{C}^{-1}$  \\
 $h_c \text{ kW/(m}^2\cdot\text{K)}$  \\
 $Q_{\text{max}} = ?$  \\
 $t_b \text{ }^\circ\text{C}^{-1}$  \\
 $Q_{\text{act}} = ?$  \\
 $\eta_{\text{fin}} = \text{ ; } ?$ 
\end{tabular}
\hfill
\begin{minipage}{0.5\text{width}}
\input{figs/p4e1.latex}
\end{minipage}

\begin{equation}
L_c = L + 0.5 \cdot t
\end{equation}

\begin{equation}
A_c = 2 \cdot \left( w \cdot L_c \right)
";\;\;\;\;\;\label{p4e1e1}"
A_c \cdot \text{m}^{-2}
";\;\;\;\;\;\;"
A_c \cdot \text{cm}^{-2}
\end{equation}

\begin{equation}
Q_{\text{max}} = h_c \cdot A_c \cdot \left( t_b - t_e \right)
";\;\;\;\;\;\;"
Q_{\text{max}} \cdot \text{kW}^{-1}
\end{equation}

\begin{equation}
A_m = L_c \cdot t
";\;\;\;\;\;\;"
A_m \cdot \text{m}^{-2}
";\;\;\;\;\;\;"
A_m \cdot \text{cm}^{-2}
\end{equation}

\begin{equation}
mL_c = \sqrt{\frac{2 \cdot h_c \cdot k \cdot A_m}{L_c^{1.5}}}
";\;\;\;\;\;\label{p4e1e1}"
mL_c
\end{equation}

```

```

\begin{equation}
\tanh(mL_c)
\end{equation}
Where
\begin{equation}"
\tanh(x)=\frac{e^{\{x\}} -e^{\{-x\}}}{e^{\{x\}} +e^{\{-x\}}}
\label{p4e1e}
"
\end{equation}

\begin{equation}
\eta_{fin}:=\frac{\tanh(mL_c)}{mL_c}
";\;\;\;\label{p4e1e}"
\eta_{fin}
";\;\;\;\;\;"
\eta_{fin}\cdot \operatorname{percent}
\end{equation}

\begin{equation}
Q_{act}:=\eta_{fin}\cdot Q_{max}
";\;\;\;\;\label{p4e1e}"
Q_{act}\cdot \operatorname{okW}
\end{equation}

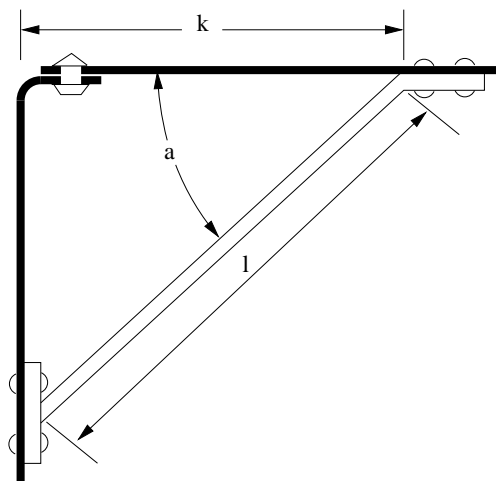
```

This following example based on Internationa Library of Technology book printed 1908 – page 50

### 1.5 Stress – a simply example

Find the diameter of a diagonal stay that supports an area  $a_1 := 6.5 \cdot \text{inches}$  by  $a_2 := 8 \cdot \text{inches}$  against a steam pressure of  $p := 90 \cdot \text{pound/inch}^2$ , the angle between the stay and the shell being  $a := 25 \cdot \text{degrees}$  and the safe tensile stress being  $T_s := 7000 \cdot \text{pound/inch}^2$ .

#### 1.5.1 Calculation



$A_s := a_1 \cdot a_2$  – area .....  $A_s \cdot (\text{inch})^{-2} = 52.0$   
 By the foregoing formula

$$d := 1.13 \cdot \sqrt{\frac{A_s \cdot p}{T_s \cdot \cos(a)}}; \quad d \cdot \text{inch}^{-1} = 0.970542357867$$

Roughly this diameter is  $d \cdot \text{inch}^{-1} = 1.0$  or  $d \cdot \text{mm}^{-1} = 25.4$  and this diameter in milimeters roughly is  $d \cdot (\text{mm})^{-1} = 25.0$ .

### 1.5.2 Source – str-calc.tex

This following example based on  
<http://books.google.com/books?id=Nh9WAAAAAAAJ&pg=RA4-PA57&dq=hand+calculation+rod#v=onepage&q=example&f=false>  
 Internationa Library of Technology book printed 1908 -- page 50

```
\Task{Stress -- a simply example}

Find the diameter of a diagonal stay that supports an area
$a_1:=6.5\cdot \text{inch}\text{es}$ by $a_2:=8\cdot \text{inch}\text{es}$
against a steam pressure of $p:=90\cdot \text{pound}/\text{inch}^2$, "$
the angle between the stay and the shell being $a:=25\cdot \text{degrees}$
and the safe tensile stress being $T_s:=7\ 000\cdot \text{pound}/\text{inch}^2$."$

\Calculation{}

%$k:=24\cdot \text{inch}$ $l:=k\cdot \cos(a)$ $l:=21.75\cdot \text{inch}$

\begin{figure}[htp] \centering
\includegraphics[width=0.4\textwidth, angle=-90]{figs/diagonal}
\end{figure}
\noindent
$A_s:=a_1\cdot a_2$ -- area \dotfill $A_s\cdot \left( \text{inch}\text{right} \right)^{-2}$\
By the foregoing formula
\[ d:=1.13\cdot \sqrt{\frac{A_s\cdot p}{T_s\cdot \cos(a)}}
";\;\;\;"
d\cdot \text{inch}^{-1} \]
Roughly this diameter is
%begin enviroment that is not printed in \CalcTeX\ but is calculated by \CalcTeX\
\texblanche{
%$d_{\text{base}}:=d$
$d:=\text{round}(d\cdot \text{inch}^{-1},0)\cdot \text{inch}$ % round d value up to 1 times 100 of inch
% \hfill $d_{\text{inch}}\cdot \text{inch}^{-1}$
}
%end of texblanche enviroment
$d \cdot \text{inch}^{-1}$ or
$d \cdot \text{mm}^{-1}$
\texblanche{%begin of \texblanche enviroment
$d:=\text{round}(d,3)$ %<-- round of d value
}% end of \texblanche enviroment
and this diameter in milimeters roughly is
\texblanche{%begin of \texblanche enviroment
$d:=\text{round}(d,3)$ %<-- round of d value up to 10-3 \times meter
}% end of
$d\cdot \left( \text{mm}\text{right} \right)^{-1}$."$
```